Waterloo



M. Bahrami J. R. Culham M. M. Yovanovich G. E. Schneider

Department of Mechanical Engineering Microelectronics Heat Transfer Laboratory University of Waterloo Waterloo, ON, Canada

CONTENTS



- introduction
- problem statement
- geometrical analysis
- mechanical analysis
- microcontacts deformation modes
- thermal analysis
- comparison between TCR models and experimental data
- summary and conclusions

INTRODUCTION



- conduction (microcontacts)
- conduction (interstitial fluid)
- radiation across the gap
- two resistances in series represent TCR in a vacuum, many researchers assumed

$$R_j = R_{mic} + R_{mac}$$

- spherical rough contact includes two problems:
 - micro scale problem
 - macro scale problem
- macrocontact area: region where microcontacts are distributed



PROBLEM STATEMENT





ROUGH SURFACE PARAMETERS

- all solid surfaces are rough
- Gaussian rough surface
- approximate estimations of *m*

$$\sigma = R_q = \sqrt{\frac{1}{L} \int_0^L z^2(x) dx}$$

$$m = \frac{1}{L} \int_0^L \left| \frac{dz(x)}{dx} \right| dx$$



Review of Thermal Joint Resistance Models For Non-Conforming Rough Surfaces in a Vacuum 2003 ASME Summer Heat Transfer Conference – Las Vegas, Nevada, July 21 - 23

EQUIVALENT ROUGH SURFACE

- equivalent surface circumvents problem of misalignment of contacting peaks, Francis (1977)
- equivalent surface of two Gaussian surfaces is itself Gaussian, Greenwood (1967)
- equivalent surface will be in general less anisotropic than the two contacting surfaces, Francis (1977)

$$\sigma = \sqrt{\sigma_1^2 + \sigma_2^2}$$
 and $m = \sqrt{m_1^2 + m_2^2}$







GEOMETRICAL MODELING



MICROHARDNESS

Hegazy (1985)

• effective microhardness is significantly greater than the bulk hardness

• microhardness decreases with increasing depth of the indenter until bulk hardness is obtained

Sridhar (1994)

• suggested empirical relations to estimate Vickers microhardness coefficients using bulk hardness

$$c_{1} = H_{BGM}(4.0 - 5.77\kappa + 4.0\kappa^{2} - 0.61\kappa^{3})$$

$$c_{2} = -0.57 + 0.82\kappa - 0.41\kappa^{2} + 0.06\kappa^{3}$$

$$\kappa = H_{B}/H_{BGM} \quad 0.41 \le \kappa \le 2.39$$



$$H_v = c_1 (d'_v)^{c_2}$$
$$d'_v = d_v / d_0$$

Review of Thermal Joint Resistance Models For Non-Conforming Rough Surfaces in a Vacuum 2003 ASME Summer Heat Transfer Conference – Las Vegas, Nevada, July 21 - 23

Waterloo



MECHANICAL ANALYSIS



MACROCONTACT PROBLEM



Hertz (1881) (elastic contact of smooth spheres)

- each body is modeled as elastic half-space loaded over small contact region
- strains are small; surfaces are frictionless
- pressure distribution assumed

$$P(r) = P_0 \sqrt{1 - (r/a_{Hz})^2}$$

Elastoplastic and Fully Plastic [Cavity model of Johnson (1985)]

 when yield point is exceeded plastic zone is small and fully contained by material which remains elastic

• contact load must be increased about 400 times from initial yielding to fully plastic flow, elastoplastic transition region is very long

- when the plastic deformation is severe, elastic deformation may be neglected
- Hardy et al. (1971) showed plastic flow leads to "flattening" of pressure distribution

MICROCONTACT



Common Assumptions

- contacting surfaces are rough, isotropic, with Gaussian asperity distribution
- microcontact are independent; interfacial force on microcontact acts normally (no friction)

 deformation mechanics (stress and displacement) are uniquely determined by shape of equivalent surface

Plastic Models

• Abott and Firestone (1933) model assumed asperities are "flattened" or, equivalently penetrate into the smooth surface without any change in shape of the part of surfaces not yet in contact

$$A_r/A_a = P_m/H_{mic}$$

• Tsukizoe and Kisakado (1965) and Cooper et al. (1969) derived relations for size and number of microcontacts

Review of Thermal Joint Resistance Models For Non-Conforming Rough Surfaces in a Vacuum 2003 ASME Summer Heat Transfer Conference – Las Vegas, Nevada, July 21 - 23

MICROCONTACT (CONT.'D)

Elastic Models, GW (1966)

- all summits have same radius of curvature; with Gaussian distribution
- distribution of summit heights is same as heights standard deviation, i.e., $\sigma_s = \sigma$
- summits deform elastically and Hertz theory applied for each individual summit

Elastoplastic Models

- Chang et al. (1987)
- Zhao et al. (2000)





MICROCONTACTS DEFORMATIONS



Plasticity Index

• GW "most surfaces have plasticity indices larger than 1.0, except for very smooth surfaces, asperities will flow plastically under the lightest loads"

$$\gamma_{GW} = (E'/H) \sqrt{\sigma/\beta}$$

• Mikic (1974) reported mode of deformation, as stated by GW, depends only on material properties and shape of asperities, and it is not sensitive to pressure level

$$\gamma_{Mikic} = H_{mic}/E'm$$

$$\lambda = Y/\sqrt{2}\sigma$$

Model	a'_s	n'_s	F'
GW	$\sqrt{\exp(-\lambda^2)/\operatorname{erfc}(\lambda)} - \sqrt{\pi} \lambda$	$\operatorname{erfc}(\lambda)$	$\sqrt{\lambda} \exp\left(-\frac{\lambda^2}{2}\right) \left[(1+2\lambda^2) K_{\frac{1}{4}}\left(\frac{\lambda^2}{2}\right) - 2\lambda^2 K_{\frac{3}{4}}\left(\frac{\lambda^2}{2}\right) \right]$
ТК	1/λ	$\lambda \exp(-\lambda^2)$	$\exp(-\lambda^2)/\lambda$
CMY	$\exp(\lambda^2)\operatorname{erfc}(\lambda)$	$\exp(-2\lambda^2)/\operatorname{erfc}(\lambda)$	$\operatorname{erfc}(\lambda)$



ELASTIC, PLASTIC MODEL TRENDS



Review of Thermal Joint Resistance Models For Non-Conforming Rough Surfaces in a Vacuum 2003 ASME Summer Heat Transfer Conference – Las Vegas, Nevada, July 21 - 23

14

THERMAL ANALYSIS



Common Assumptions

- contacting solids are isotropic; thermophysical properties are constant
- contacting solids are thick relative to roughness or waviness scales
- surfaces are clean; contact is static
- radiation heat transfer is negligible
- microcontacts are circular; microcontacts are isothermal and flat
- steady-state heat transfer at microcontacts

SPREADING/CONSTRICTION RESISTANCE

TCR models assume a number of heat channels exist within macrocontact area

1. Heat Source on a Half-space

classical steady-state solutions are available for two boundary conditions;

- isothermal
- isoflux
- difference between isoflux and isothermal heat sources is 8%

$$R_{s,isoflux} = 1.08R_{s,isothermal}$$



$$R_{s,isothermal} = 1/(4ka)$$

SPREADING RESISTANCE 2







NON-CONFORMING ROUGH TCR MODELS

Clausing and Chao (1963)

• plastic microcontacts; H_{mic} corrected by empirical factor to account for elastic deformation of asperities

• identical microcontacts uniformly distributed, in triangular array, over macrocontact region

• microcontacts considered as isothermal circular heat sources on a half-space

• average size of microcontacts a_s is independent of load and it is of same order of magnitude as surface roughness, i.e., $a_s = \sigma$

 neglecting effect of roughness on macrocontact, radius of macrocontact, a_L, obtained from Hertz theory



$$R_j = R_{mic} + R_{mac}$$

NON-CONFORMING ROUGH TCR MODELS 2



- Mikic and Rohsenow (1966) studied TCR for various types of surface waviness and conditions
- **Kitscha (1982)** and **Fisher (1985)** developed models similar to Clausing and Chao's model and experimentally verified their models for relatively small radii of curvature and different levels of roughness
- **Burde (1977)** derived expressions for size distribution, and number of microcontacts, which described the increase in macrocontact radius for increasing roughness
- Lambert (1995) studied TCR of two rough spheres in a vacuum



COMPARISON WITH DATA

- Clausing and Chao (1963)
- Yovanovich (1982)
- Lambert (1995)
- Yovanovich (1986) elasto-constriction approximation

Parameters		
$57.3 \le E' \le 114.0$ (GPa)		
$16.6 \le k_s \le 75.8 (W/mK)$		
$0.12 \le \sigma \le 13.94 \ (\mu m)$		
$0.04 \le m \le 0.34$ (-)		
$0.013 \le \rho \lessapprox 120 \ (m)$		

Ref.	Researcher	Specimen Material(s)	
A	Antonetti (1983)	Ni 200	
В	Burde (1977)	SPS 245, Carbon Steel	
F	Fisher (1985)	Ni 200, Carbon Steel	
Н	Hegazy (1985)	Ni 200SS 304Zircaloy4Zr-2.5% wt Nb	
K	Kitscha (1982)	Steel 1020,Carbon Steel	
М	Milanez et al. (2003)	SS 304	

TCR LIMITS



Review of Thermal Joint Resistance Models For Non-Conforming Rough Surfaces in a Vacuum 2003 ASME Summer Heat Transfer Conference – Las Vegas, Nevada, July 21 - 23

21

Waterloo



ELASTOCONSTRICTION LIMIT



Review of Thermal Joint Resistance Models For Non-Conforming Rough Surfaces in a Vacuum 2003 ASME Summer Heat Transfer Conference – Las Vegas, Nevada, July 21 - 23

22



CONFORMING ROUGH LIMIT



Review of Thermal Joint Resistance Models For Non-Conforming Rough Surfaces in a Vacuum 2003 ASME Summer Heat Transfer Conference – Las Vegas, Nevada, July 21 - 23

23

SUMMARY AND CONCLUSIONS



- TCR modeling consists of three analyses: geometrical, mechanical, and thermal; each one includes a macro and micro part
- recommended empirical correlations, *m*, were summarized and compared with experimental data. Uncertainty of correlations is high
- a set of scale relationships were derived for contact parameters for GW elastic, CMY and TK plastic conforming rough models. It was graphically shown that their trends are similar
- trends of contacting rough surfaces was determined essentially by surface statistical characteristics. Also combination of plastic and elastic modes would introduce no new features

SUMMARY AND CONCLUSIONS 2



- existing correlations for flux tube resistance were compared; it was shown all correlations show good agreement for the applicable range
- experimental data of many researchers were summarized and grouped into two limiting cases:
 - conforming rough $\rho \rightarrow \infty$
 - elasto-constriction $\sigma \rightarrow 0$

 data were non-dimensionalized and compared with TCR models at limiting cases

• no existing theoretical model covers both limiting cases

Waterloo

ACKNOWLEDGEMENTS



 Natural Sciences and Engineering Research Council of Canada (NSERC)

• The Center for Microelectronics Assembly and Packaging (CMAP)