
Thermal Contact Resistance of Non-Conforming Rough Surfaces

Part 2: Thermal Model

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CONTENTS

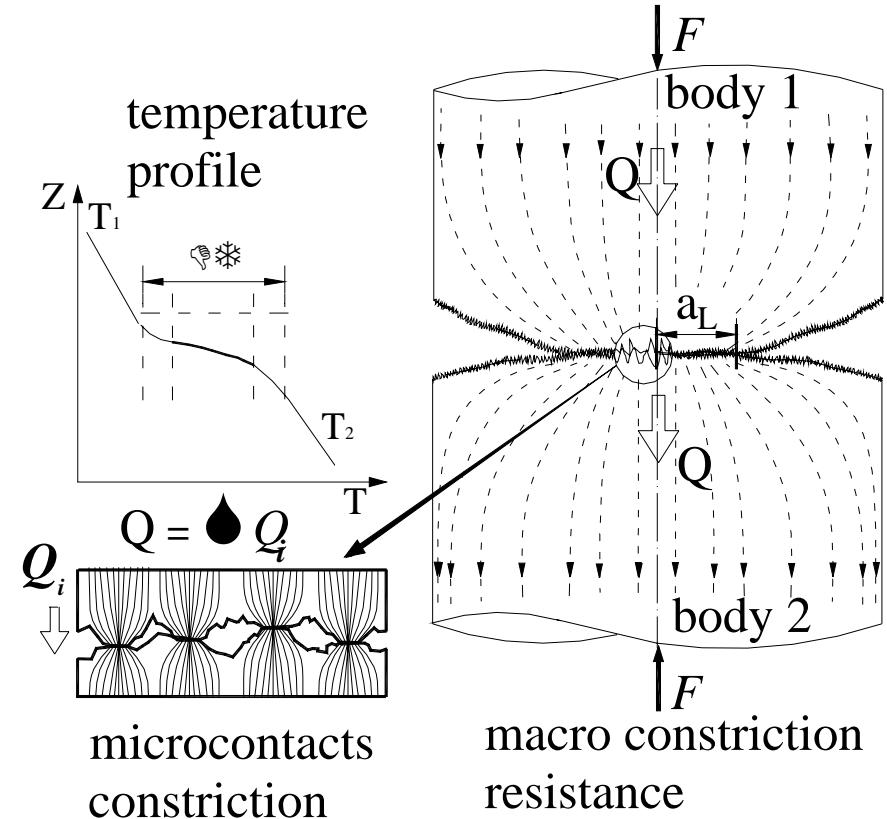
- introduction
 - objectives
 - literature review
 - present model
 - numerical approach and results
 - parametric study
 - alternative approach (correlations)
 - comparison with experimental data
 - summary and conclusions
-

INTRODUCTION

- conduction (microcontacts)
- conduction (interstitial fluid)
- radiation across the gap
- two sets of resistances in series represent TCR in a vacuum
- many researchers assumed

$$R_j \quad R_{mic} \quad R_{mac}$$

- Bahrami et al. (2003) reviewed existing non-conforming rough models



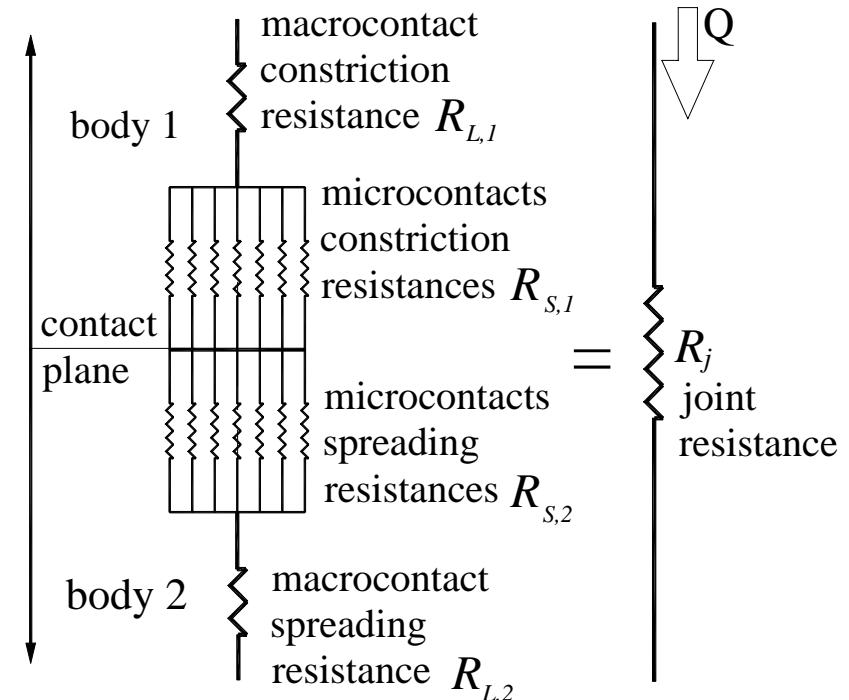
THERMAL RESISTANCE NETWORK

thermal resistance is defined as,

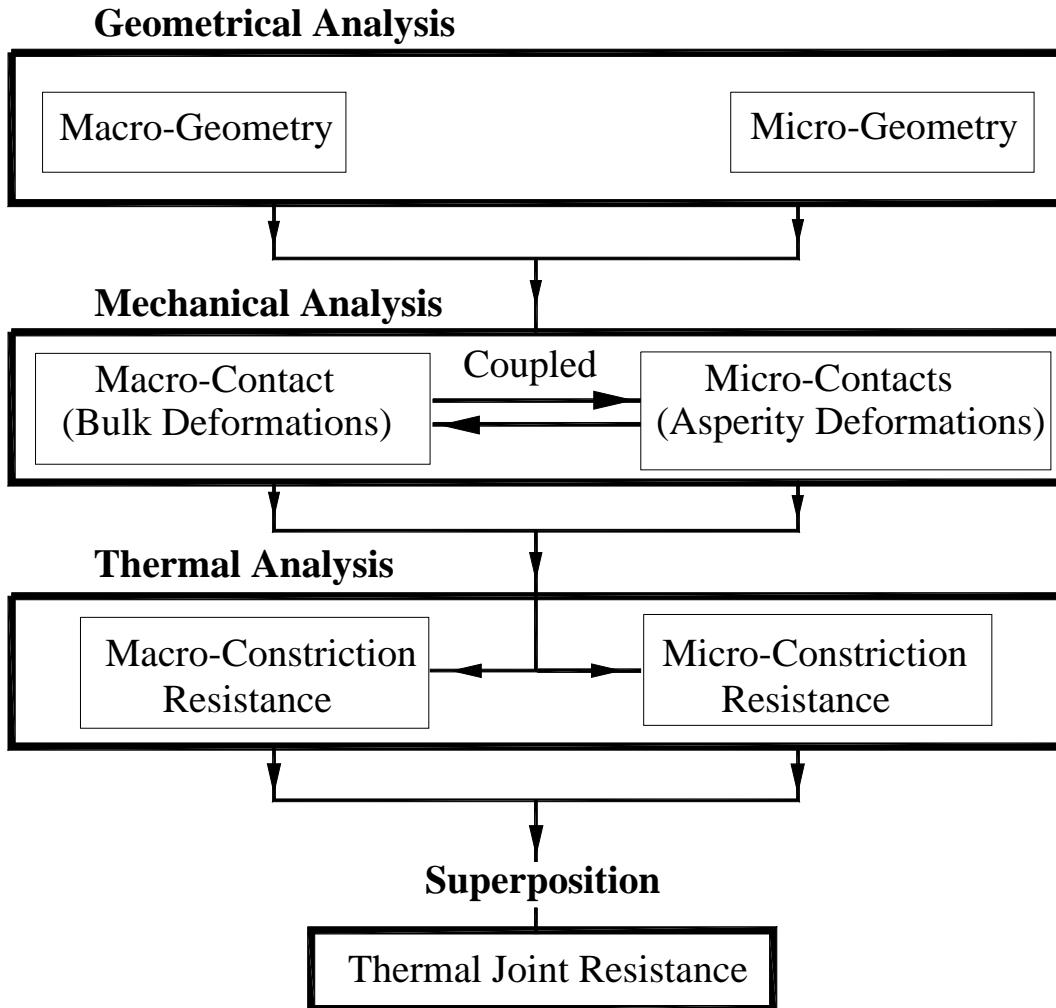
$$R \quad T/Q$$

$$R_j \quad R_{L,1} \quad R_{s,1} \quad R_{s,2} \quad R_{L,2}$$

$$\left(\frac{1}{R_s} \right)_{1,2} \quad \begin{pmatrix} n_s & \frac{1}{R_{s,i}} \\ i & 1 \end{pmatrix}_{1,2}$$



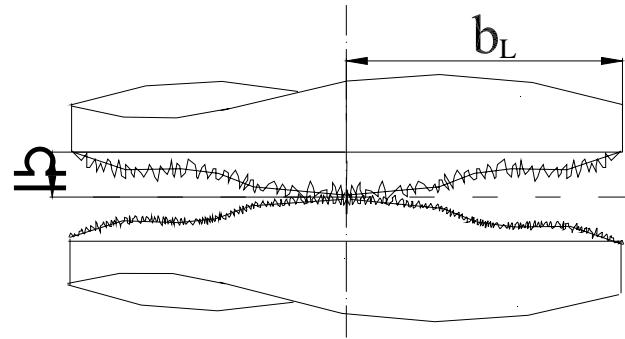
TCR PROBLEM STATEMENT



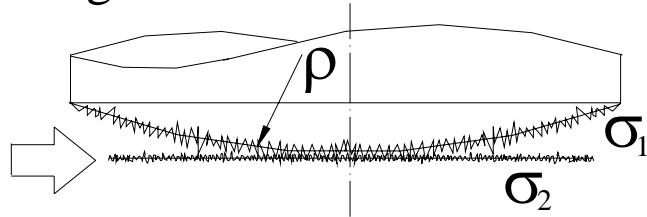
OBJECTIVES

- develop analytical TCR model for entire range of contacts:
 - conforming rough
 - elastoconstriction
 - transition region
- study effects of input parameters on TCR
- derive simple correlations for determining TCR

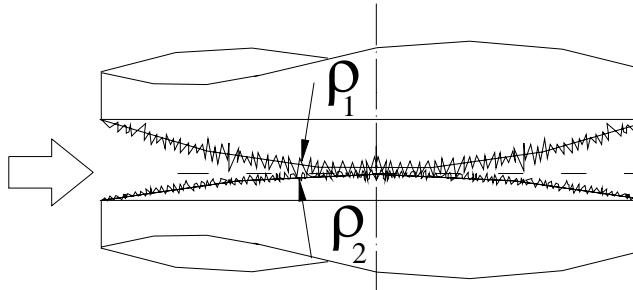
GEOMETRICAL MODELING



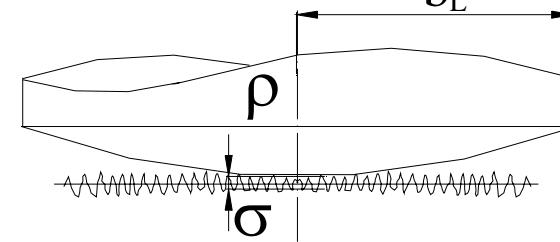
a) contact of non-conforming rough surfaces



c) rough sphere-flat contact,
effective radius of curvature



b) contact of two rough spherical segments b_L



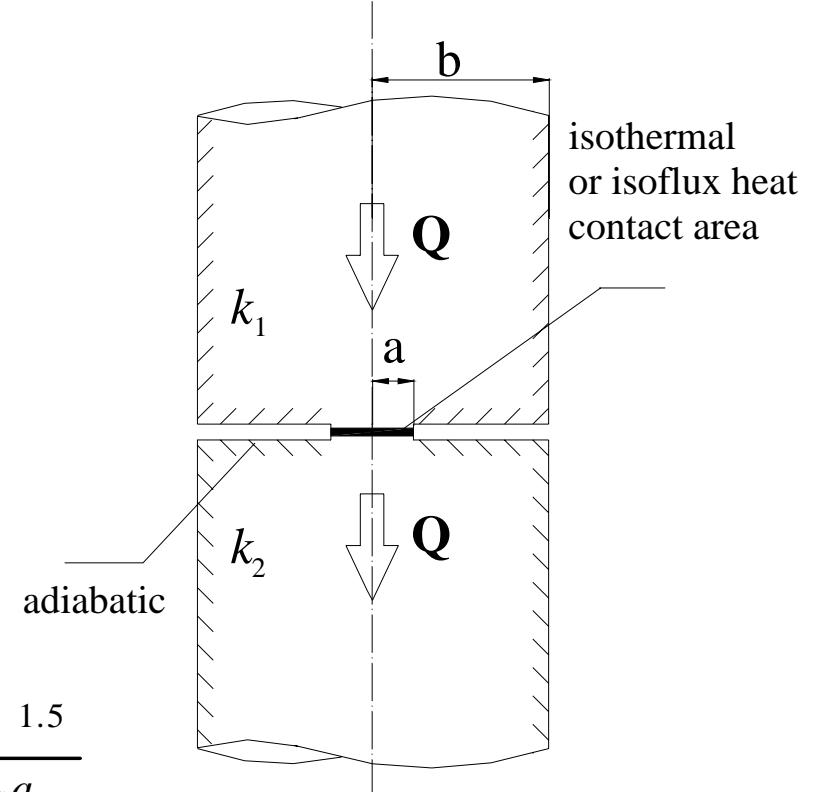
d) equivalent sphere-flat contact,
effective radius and roughness

$$\frac{b_L^2}{2}$$

FLUX TUBE SOLUTION

- inside macrocontact area a number of parallel cylindrical heat channels exist
- basic element for macro and micro thermal analysis
- Cooper et al. (1969)

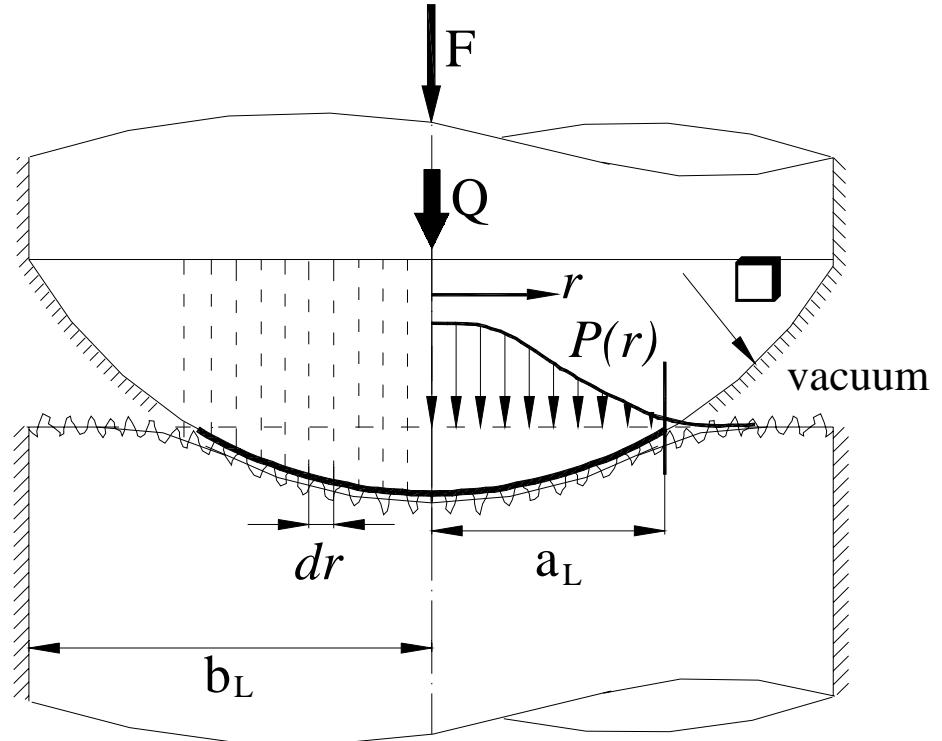
$$R_{\text{flux tube } 1} = R_{\text{flux tube } 2} = \frac{1 - \frac{1.5}{2k_s a}}{\frac{2k_s a}{k_1 k_2}}$$



$$\frac{a}{b} = \frac{k_s}{k_1 k_2}$$

PRESENT MODEL (ASSUMPTIONS)

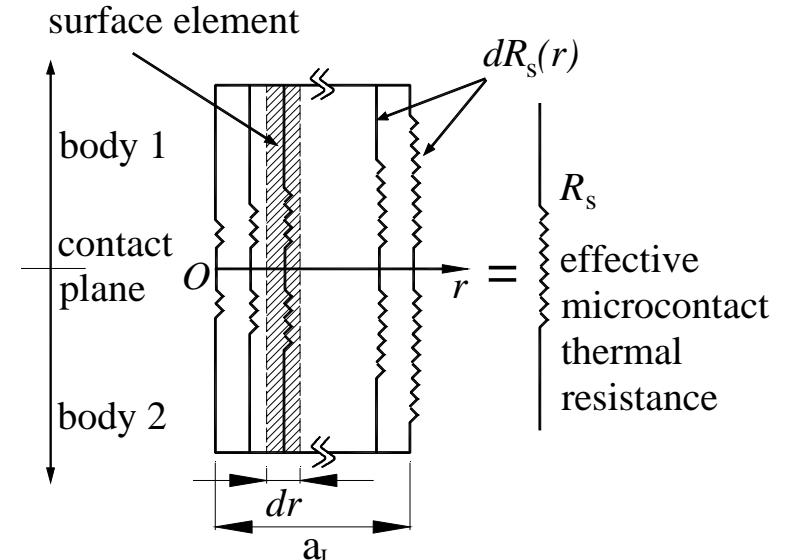
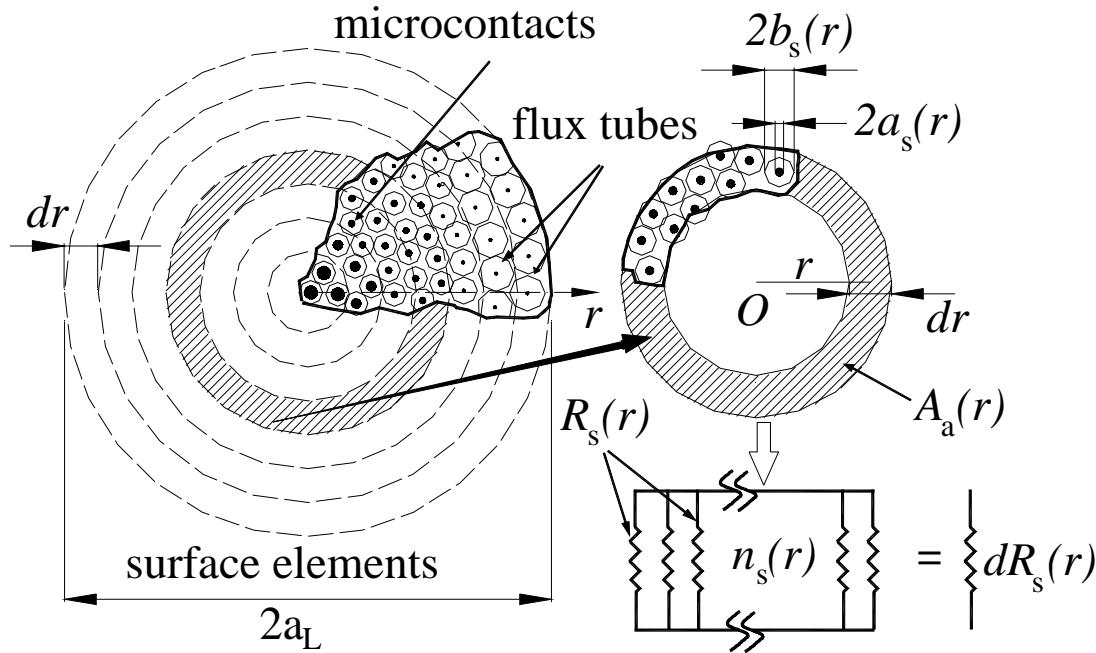
- solids are isotropic
- radiation heat transfer is negligible
- microcontacts are circular and steady-state heat transfer
- isothermal microcontacts
- microcontacts are flat
- surfaces are clean



$$R_j \quad R_L \quad R_s$$

$$R_L \quad \frac{1-a_L/b_L}{2k_s a_L}^{3/2}$$

MICROCONTACTS RESISTANCE



$$R_s \ r = \frac{r}{2k_s a_s \ r}$$

$$r = \sqrt{\frac{A_r \ r}{A_a \ r}} = \sqrt{\frac{1}{2} \operatorname{erfc} \frac{r}{\sqrt{2}}}$$

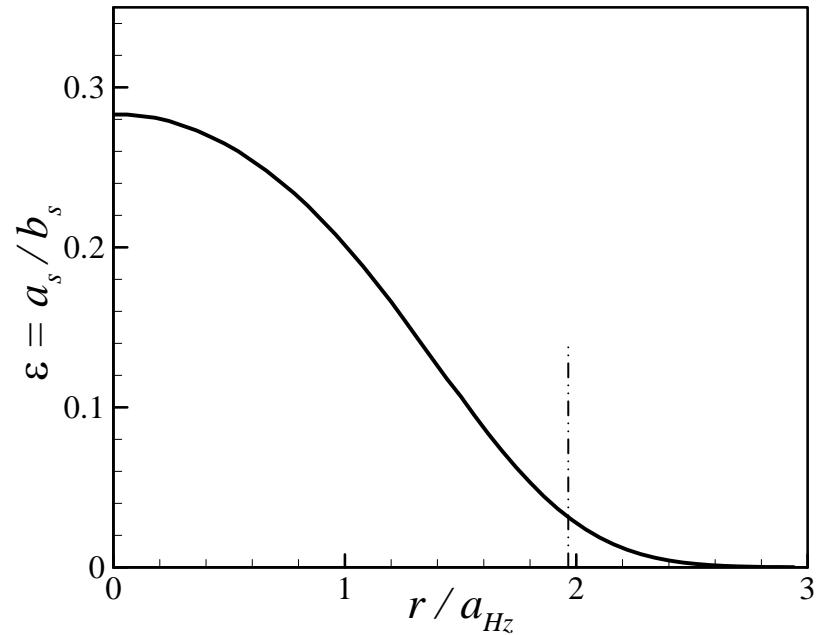
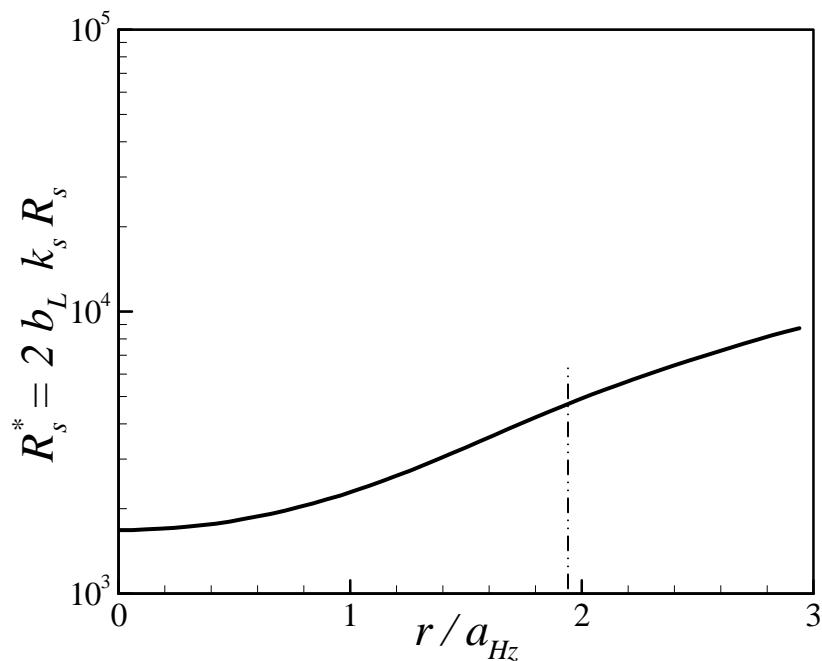
$$n_s = \frac{1}{16} \left(\frac{m}{a_L} \right)^2 \frac{\exp[-2 \left(\frac{r}{a_L} \right)^2]}{\operatorname{erfc} \frac{r}{\sqrt{2}}} A_a$$

$$dR_s \ r = \frac{R_s \ r}{n_s \ r}$$

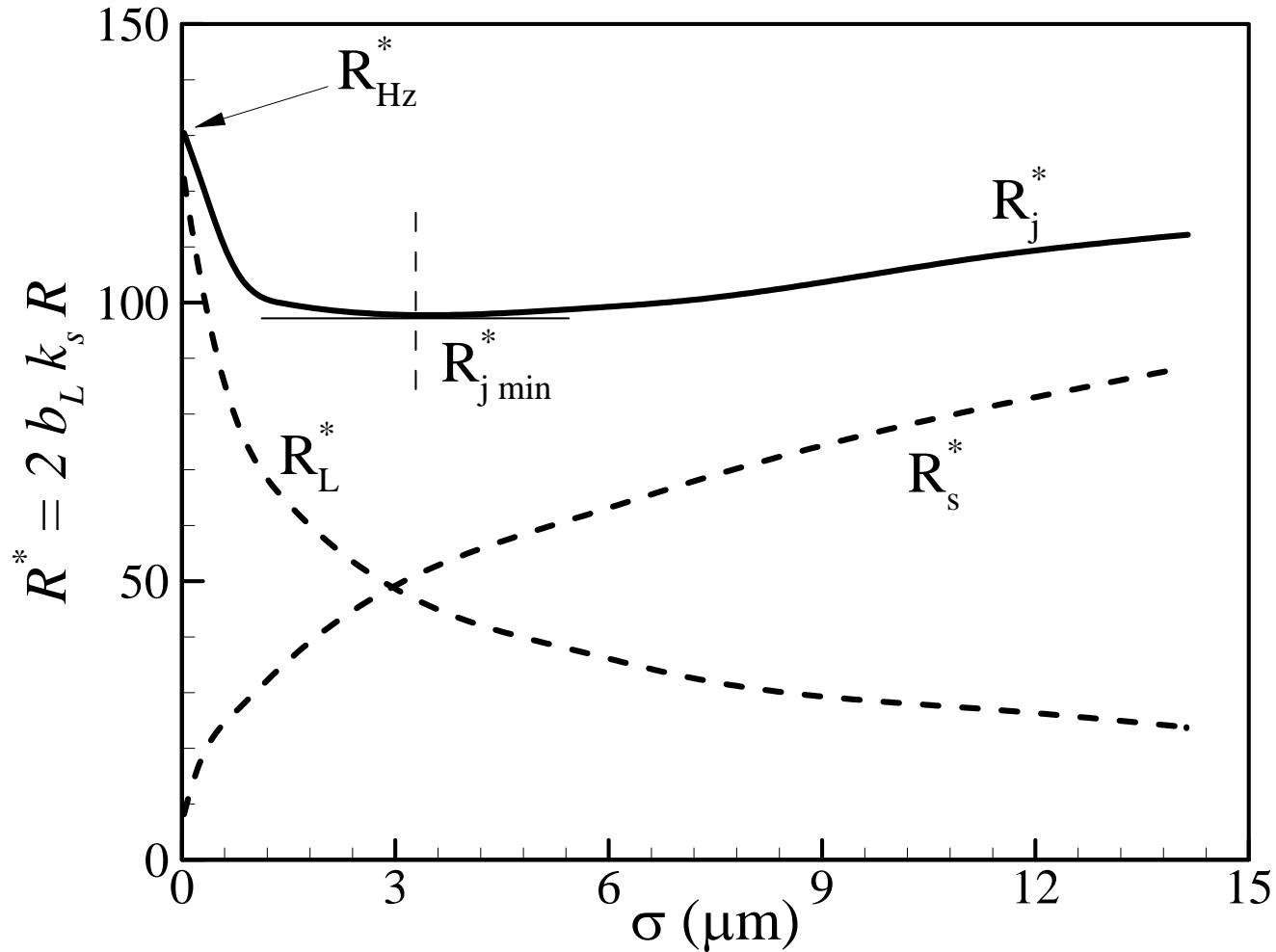
$$R_s = \frac{1}{1/dR_s \ r}$$

NUMERICAL RESULTS

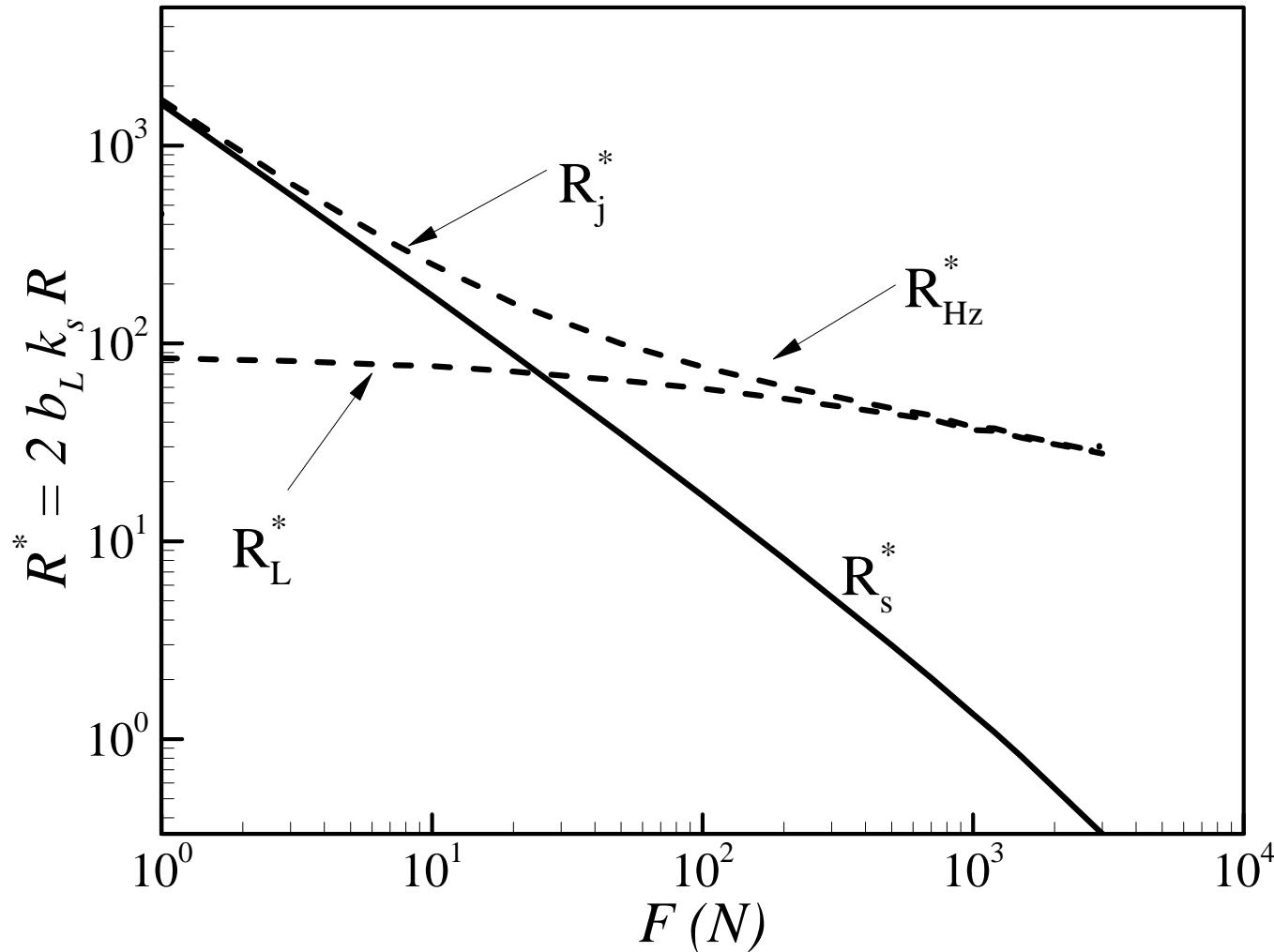
25 mm	$F \quad 50 \text{ N}$
1.41 m	$E \quad 112.1 \text{ GPa}$
$m \quad 0.107 \text{ --}$	$c_1/c_2 \quad 6.27 \text{ GPa} / -0.15 \text{ --}$
$b_L \quad 25 \text{ mm}$	$k_s \quad 16 \text{ W/mK}$



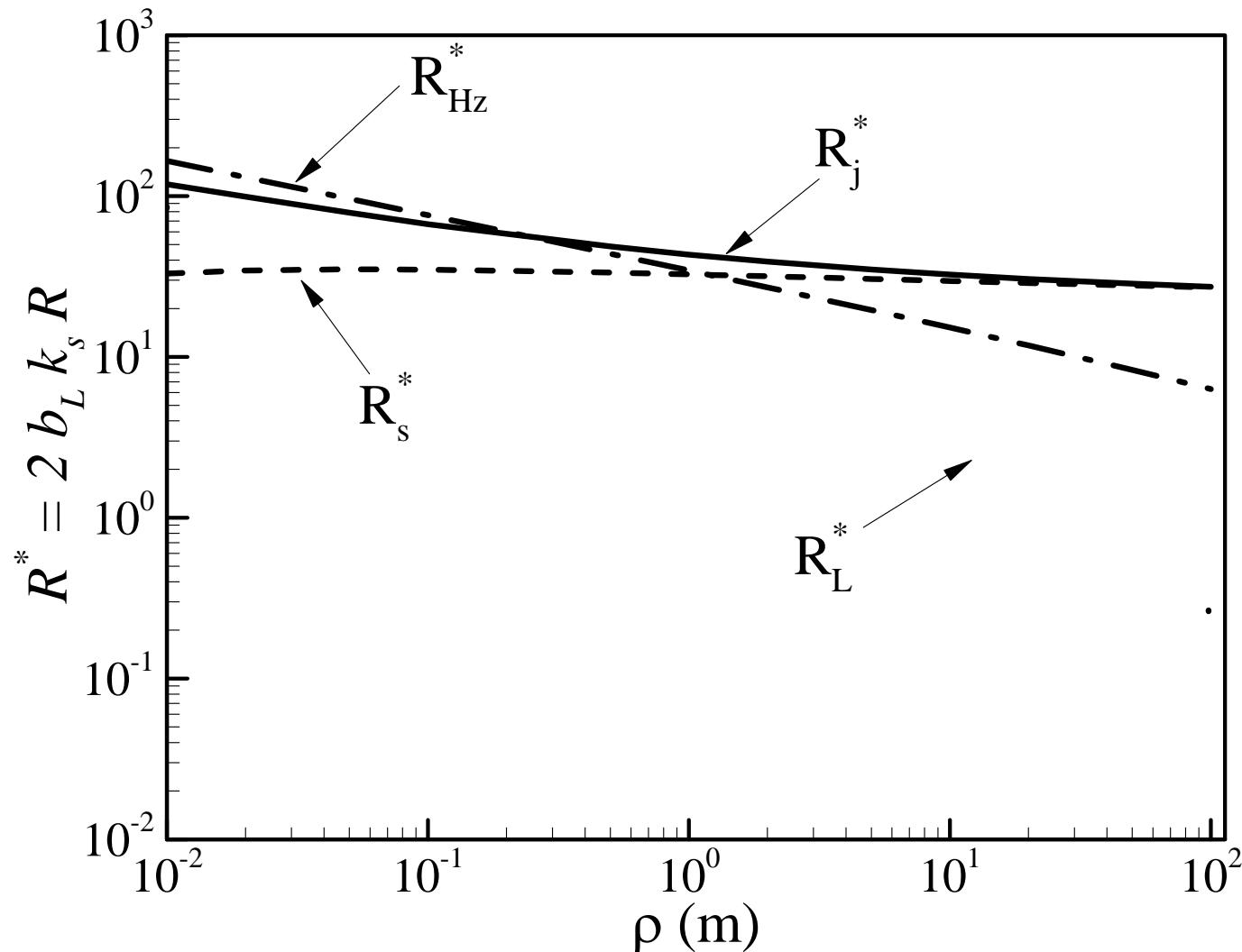
EFFECT OF ROUGHNESS



EFFECT OF LOAD



EFFECT OF RADIUS OF CURVATURE



ALTERNATIVE APPROACH

- conforming rough conductance, Yovanovich (1982)

$$h_s = 1.25k_s \frac{m}{\left(\frac{P}{H_{mic}} \right)^{0.95}}$$

- heat transfer in non-conforming rough contact:

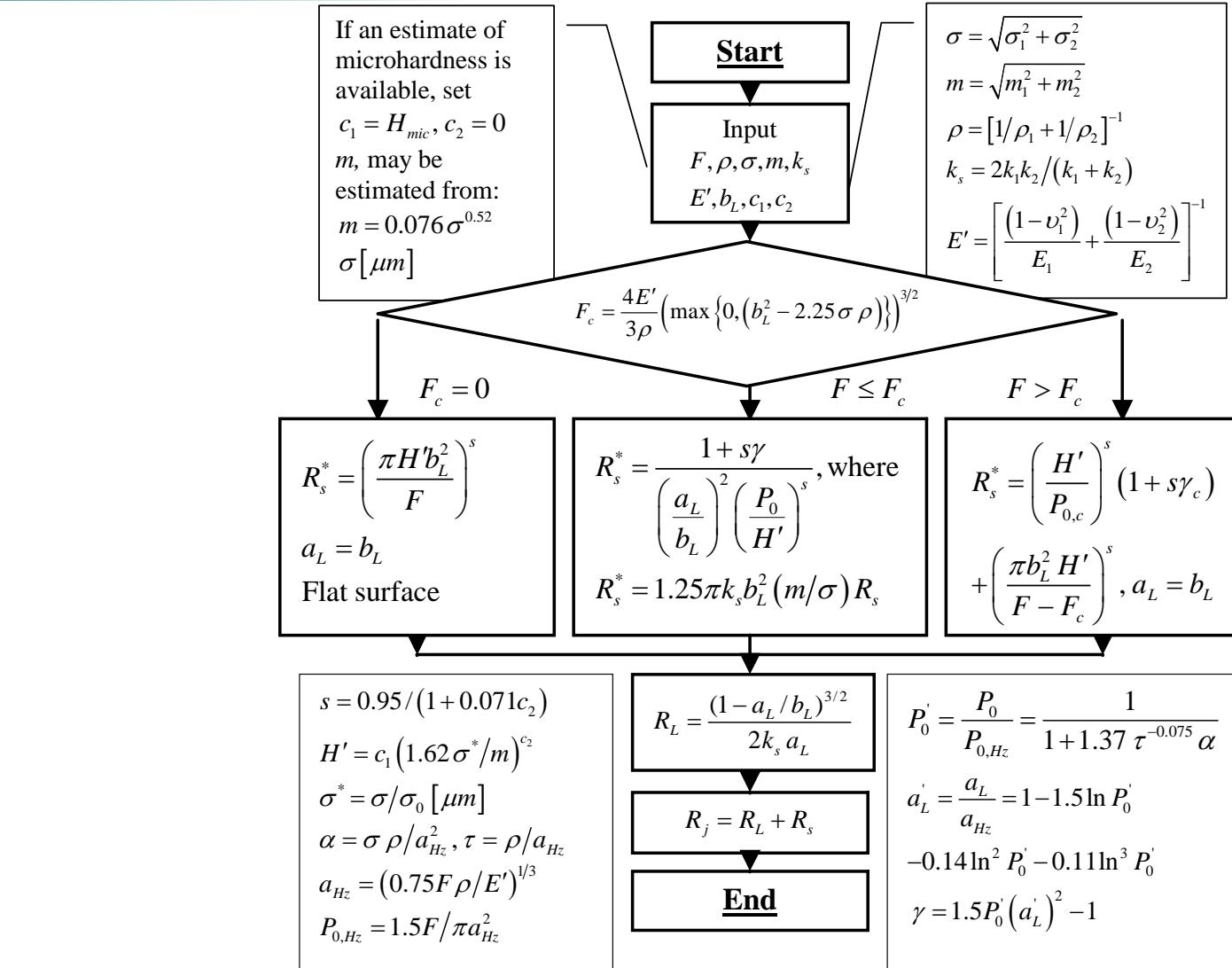
$$Q = h_s \int r T_s dA_a$$

contact plane

- using pressure distribution, Part I and $R = 1/hA_a$

$$R_s = \frac{1}{1.25} \frac{s}{m k_s a_L^2} \left(\frac{H}{P_0} \right)^s$$

PROCEDURE FOR UTILIZING THE MODEL

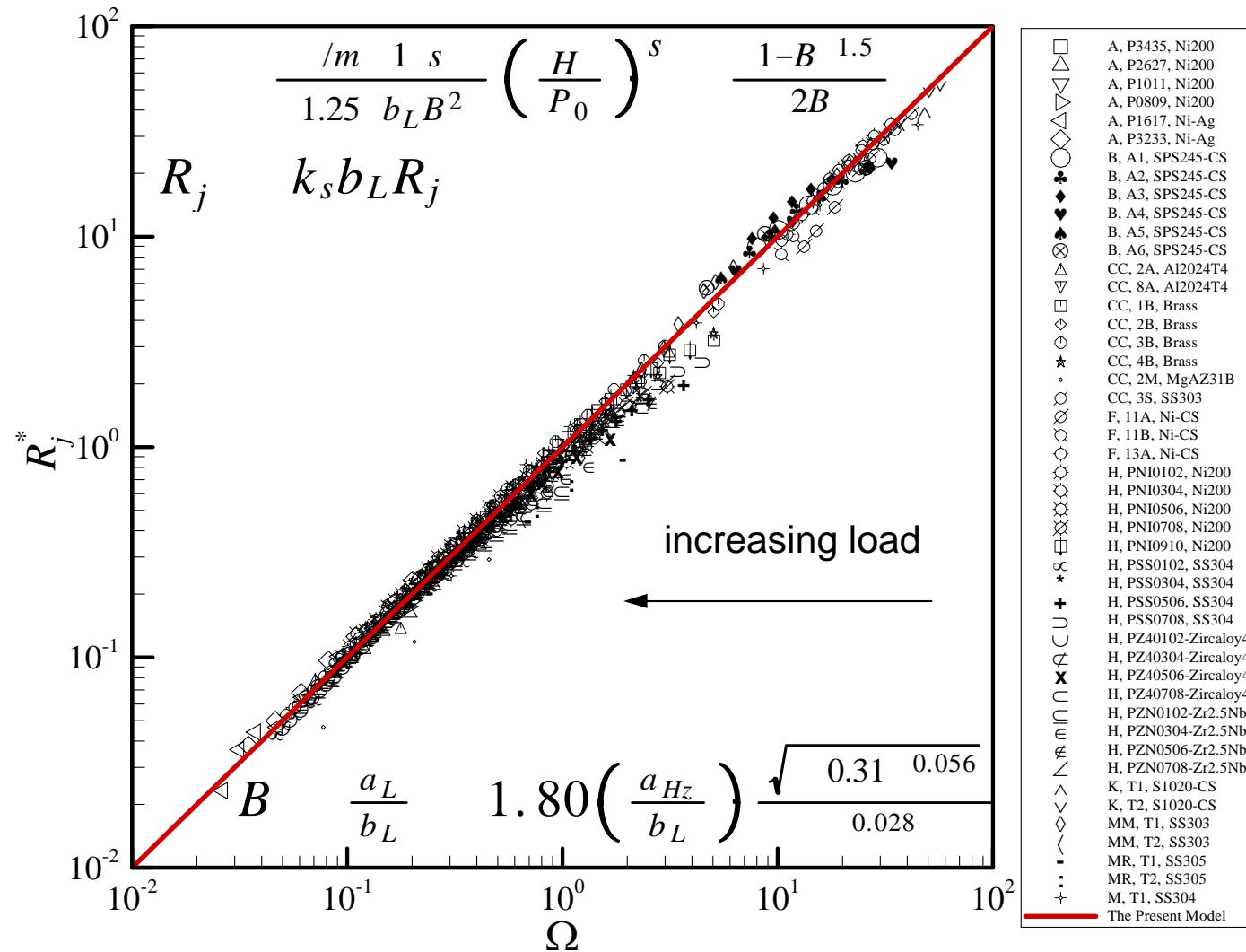


EXPERIMENTAL DATA

Parameter		
7.15	b_L	14.28 mm
25.64	E	114.0 GPa
7.72	F	16763.9 N
16.6	k_s	227.2 W/mK
0.04	m	0.34 –
0.12		13.94 m
0.013		120 m

Ref.	Researcher	Material(s)
A	Antonetti	{ Ni200 Ni200-Ag }
B	Burde	SPS 245, CS
CC	Clausing-Chao	{ Al2024 T4 Brass Anaconda Mg AZ 31B SS303 }
F	Fisher	Ni 200-Carbon Steel
H	Hegazy	{ Ni200 SS304 Zircaloy4 Zr-2.5% wt Nb }
K	Kitscha	Steel 1020-CS
MM	McMillan-Mikic	SS303
MR	Mikic-Rohsenow	SS305
M	Milanez et al.	SS304

COMPARISON WITH DATA



SUMMARY AND CONCLUSIONS

- superposition of macro and micro thermal resistance
 - effects of major parameters, i.e., roughness, load, and radius of curvature on TCR were investigated
 - for non-conforming rough contact, there is a value of surface roughness that minimizes TCR
 - at relatively large loads effect of roughness on TCR becomes negligible
 - simple correlations were derived that cover entire range of TCR
-

SUMMARY AND CONCLUSIONS 2

- model was compared with more than 700 experimental data points, collected by many researchers
- comparison includes all three regions of TCR and wide range of mechanical, thermal, and surfaces characteristics
- data include contact between dissimilar metals such as Ni200-Ag and SS-CS
- RMS relative difference between the model and the data was estimated to be approximately 11.4%

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- Natural Sciences and Engineering Research Council of Canada (NSERC)
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CONFORMING ROUGH DATA

Ref.	E		m	c ₁	-c ₂	k _s	b _L
A,P3435	112.1	8.48	.34	6.3	.26	67.1	14.3
A,P2627	112.1	1.23	.14	6.3	.26	64.5	14.3
A,P1011	112.1	4.27	.24	6.3	.26	67.7	14.3
A,P0809	112.1	4.29	.24	6.3	.26	67.2	14.3
A,P1617	63.9	4.46	.25	.39	0	100	14.3
A,P3233	63.9	8.03	.35	.39	0	100	14.3
H,NI12	112.1	3.43	.11	6.3	.26	75.3	12.5
H,NI34	112.1	4.24	.19	6.3	.26	76.0	12.5
H,NI56	112.1	9.53	.19	6.3	.26	75.9	12.5
H,NI78	112.1	13.9	.23	6.3	.26	75.7	12.5
H,NI910	112.1	0.48	.23	6.3	.26	75.8	12.5
H,SS12	112.1	2.71	.07	6.3	.23	19.2	12.5
H,SS34	112.1	5.88	.12	6.3	.23	19.1	12.5
H,SS56	112.1	10.9	.15	6.3	.23	18.9	12.5
H,SS78	112.1	0.61	.19	6.3	.23	18.9	12.5
H,Z412	57.3	2.75	.05	3.3	.15	16.6	12.5
H,Z434	57.3	3.14	.15	3.3	.15	17.5	12.5
H,Z456	57.3	7.92	.13	3.3	.15	18.6	12.5
H,Z478	57.3	0.92	.21	3.3	.15	18.6	12.5
H,ZN12	57.3	2.50	.08	5.9	.27	21.3	12.5
H,ZN34	57.3	5.99	.16	5.9	.27	21.2	12.5
H,ZN56	57.3	5.99	.18	5.9	.27	21.2	12.5
H,ZN78	57.3	8.81	.20	5.9	.27	21.2	12.5
M,SS1	113.8	0.72	.04	6.3	.23	18.8	12.5

ROUGH SPHERE-FLAT DATA

Ref.	E	/m		$c_1/ - c_2$	k_s	b_L
B,A-1	114.0	0.63/.04	.013	3.9/0	40.7	7.2
B,A-2	114.0	1.31/.07	.014	3.9/0	40.7	7.2
B,A-3	114.0	2.44/.22	.014	3.9/0	40.7	7.2
B,A-4	114.0	2.56/.08	.019	4.4/0	40.7	7.2
B,A-5	114.0	2.59/.10	.025	4.4/0	40.7	7.2
B,A-6	114.0	2.58/.10	.038	4.4/0	40.7	7.2
CC,2A	38.66	0.42/-	14.0	1.6/.04	141	12.7
CC,8A	38.66	2.26/-	14.7	1.6/.04	141	12.7
CC,1B	49.62	0.47/-	3.87	3.0/.17	125	12.7
CC,2B	49.62	0.51/-	4.07	3.0/.17	125	12.7
CC,3B	49.62	0.51/-	3.34	3.0/.17	102	12.7
CC,4B	49.62	0.51/-	4.07	3.0/.17	125	12.7
CC,3S	113.7	0.11/-	21.2	4.6/.13	17.8	12.7
CC,2M	25.64	0.11/-	30.3	.41/0	96	12.7
F,11A	113.1	0.12/-	.019	4.0/0	57.9	12.5
F,11B	113.1	0.12/-	.038	4.0/0	57.9	12.5
F,13A	113.1	0.06/-	.038	4.0/0	58.1	12.5
K,T1	113.8	0.76/-	.014	4.0/0	51.4	12.7
K,T2	113.8	0.13/-	.014	4.0/0	51.4	12.7
MM,T1	113.7	2.7/.06	.128	4.0/0	17.3	12.7
MM,T2	113.7	1.75/.07	2.44	4.0/0	22	12.7
MR,T1	107.1	4.83/-	21.2	4.2/0	19.9	12.7
MR,T2	107.1	3.87/-	39.7	4.2/0	19.9	12.7

CORRELATIONS

$$F_c = \frac{4E}{3} \max(0, b_L^2 - 2.25)^{3/2}$$

$$R_s^* = \begin{cases} \left(\frac{\pi H' b_L^2}{F}\right)^s & F_c = 0 \\ \left(\frac{b_L}{a_L}\right)^2 \left(\frac{H'}{P_0}\right)^s (1 + s\gamma) & F \leq F_c \\ \left(\frac{H'}{P_{0,c}}\right)^s (1 + s\gamma_c) + \left(\frac{\pi H' b_L^2}{F - F_c}\right)^s & F \geq F_c \end{cases}$$

R_s	1.25	$b_L^2 k_s$	$m/$	R_s
s	0.95/	1	0.071	c_2
H	c_1	1.62	$/m^{c_2}$	

$$P_0 = \frac{P_0}{P_{0,Hz}} = \frac{1}{1.37^{-0.075}}$$

$$a_L = \frac{a_L}{a_{Hz}} = 1.80 \frac{\sqrt{0.31 \ 0.056}}{0.028}$$